



A computational model for large deformations of composites with a 2D soft matrix and 1D anticracks



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ABSTRACT

Anticracks (also known as rigid line inclusions) occur frequently in a variety of natural and engineered composites as very stiff and extremely sharp (almost zero-thickness) fibers or lamellae embedded in a softer matrix.

In the linear elastic regime, similarly to cracks, anticracks generate a singularity in the stress distribution around the tip. Because of this similarity, existing analytical techniques and solutions (for simple cases) can be easily translated to anticracks. However, despite their importance in many biological and engineering composites, there has been surprisingly little development of numerical methods that would account simultaneously for the presence of multiple fibers or lamellae, arbitrary loadings and nonlinear behavior of the matrix.

This paper presents the first numerical approach for rigid line inclusions, based on a meshfree scheme recently developed for multiple crack growth in elastic media. The inclusion of zero thickness is created as a crack, and a rigid motion (rotation and translation) is enforced at the anticrack faces. The equations of motion are solved according to a Total Lagrangian framework, and the matrix supposed hyperelastic.

Contrarily to available analytical solutions, the degrees of freedom of the rigid motion are determined *a posteriori* as a consequence of the (discretized) elastic equilibrium, expressed in a variational approach.

Results show that the proposed approach match well the analytical solutions and provides accurate stress intensity factors (SIFs) for relatively little computational cost. Moreover, the method can reproduce some peculiar features of the anticracks: unlike cracks, singularities also appear under compressive and parallel loads; moreover, for a certain combination of biaxial load, stress concentrations disappear.

Finally, the paper presents examples drawn from biological and engineering composites: the reorientation of one or more fibers under large strains, resulting in a smart stiffening and strengthening mechanism. Reorienting towards the direction of applied load has structural importance since reinforcements can have the most effectiveness in withstanding loads. If the matrix is compliant, the reorientation is eased.

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1. Introduction

A rigid line inclusion (RLI) is a mathematical abstraction of an extremely thin stiff inclusion dispersed within a matrix. The definition assumes the inclusion as infinitely rigid and *zero-thickness*. Kinematically, this model consists in a surface of discontinuity (a crack) where a rigid motion is imposed on all the material points belonging to the upper and lower faces of the inclusion. For this reason, some authors (Hurtado et al., 1996) refer to this model as

an *anticrack*. However, in geology this terminology indicates something different: a classical Mode I crack displacement solution with a reversed sign (Fletcher and Pollard, 1981), which in classical fracture mechanics means a violation of the non-penetrability of the crack faces. However, the justification is the dissolution and removal of material when the anticrack surfaces move toward each other, which is useful to explain triggering mechanism for snow slab avalanches (Heierli et al., 2008) or shallow earthquakes (Green et al., 1990; Burnley and Green, 1989). In RLI instead, the impenetrability is automatically imposed by a rigid motion common to both faces.

RLIs are useful to model the effects on the matrix of thin reinforcements in form of fibers, platelets, needles or rods of

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characteristic sizes much smaller than that of the embedding matrix. These reinforcements appear in many biological systems and engineered nanocomposites. For example, in biological systems (Pingle et al., 2008) like bones, teeth or nacre, the reinforcement is usually in mineralized crystal form arranged in a *staggered* disposition within a protein matrix. In calcified tissues, (Landis, 1995), these fibers influence their strength, and the overall effect is a tough nanocomposite (Ji and Gao, 2004; Pugno, 2006) produced from very *poor* materials (Fratzl and Guille, 2011). In engineered nanocomposites, RLIs appear as needle-like reinforcements (Bilotti et al., 2008, 2009, 2010), nanowhiskers (Eichhorn et al., 2010), nanoplatelets (Porwal et al., 2013b,a,c) and carbon nanotubes (Nishimura and Liu, 2004).

Many theoretical papers are available in the literature for the RLI problem, often encountered with different terminology, such as *line stiffener* or *anticrack*, owed to its resemblance with a crack. Most likely, this abundance is due to the application to RLI of already well-known techniques at that time for 2D problems: for instance, the Muskhelishvili solutions in terms of complex variable, and the Wiener–Hopf technique, previously applied for crack problems (Muskhelishvili, 1953).

Probably the first paper on RLI appeared in 1973 (Atkinson, 1973), with the term *ribbon* instead of *rigid line inclusion*. The scope of this paper was to study the response of a metallic strain measuring device in a rubber matrix. This paper presented firstly the solution for stresses in an elastic linear matrix due to a single isolated rigid *ribbon*, and secondly the solution for the *elastic ribbon*. The crack analogy is then exploited to obtain the solution for two collinear rigid inclusions, and finally, the interaction of a RLI with a free boundary. Later, Brussat and Westmann (1975) proved the correspondence between the Westergaard stress function for cracks and a stress function for RLI, and subsequently, the relation between their *stress intensity factors* (SIFs). Hasebe et al. (1984) instead proposed a rational mapping function (again taken from the elasticity of cracks) to analyze the stress state near a the tip of a crack initiated from the tip of a RLI. Wang et al. (1985) obtained the asymptotic expansion near the tip of a RLI (reported in Section 2.2 of this paper) for both stress and strain fields. Chen (1986) and later Stagni (1989) proved the path-independence of the *J-integral* around the tip of a RLI, and found that the *J-integral* for an anticrack is negative, rather than positive like in cracks. Dundurs and Markenscoff (1989) and Ballarini (1987) reported a *full-field* solution for the stresses in the matrix due to a RLI, respectively using a *weight function technique* and an *integral equation approach*, and later for a RLI at the interface of two dissimilar materials (Ballarini, 1990). Hurtado et al. (1996) introduced the term *anticrack* for RLI and *quasicracks* for elastic line inclusion: they obtained similar solutions to Atkinson (1973) starting from the Eshelby's ellipsoidal *equivalent inclusion*, for the limit to zero of the ratio between the axes.

Despite the great amount of theoretical work produced over the years, there was no attention to investigate experimentally the stress distribution near a line stiffener, until 2008, when Dal Corso et al. (2008) and Bigoni et al. (2008), and later Dal Corso and Bigoni (2009) and Noselli et al. (2010) interestingly disclosed, with photo-elasticity, the *full-field* stress state of an extremely thin and stiff inclusion made of steel embedded in a transparent epoxy matrix. They validated with their experiments some intriguing aspects of the RLI problem, already known from the analytical solutions: for instance, the appearance of a *square root* singularity also for tensile loading parallel to the stiffener.

With the field of nanocomposites in rapid growth, it becomes of paramount importance to develop numerical methods that implement RLI models that could be used by materials scientists and engineers to investigate the toughness properties of both natural and man-made composites, or to imitate artificially the

hierarchical structures present in nature. This topic seems to have been overlooked by researchers in numerical methods, with almost absent literature in this field. It is worth to acknowledge the significant contributions of Radtke et al. (2010, 2011) where they employ a Partition of Unity Finite Element Method (PUFEM) to introduce short thin fibers in a cementitious matrix as a *tunneling crack* with a finite very short thickness, not zero. The *tunnel* is introduced as a two-dimensional *Heaviside* enrichment (1 inside the fiber, 0 otherwise) over the span of the fiber. Instead, we introduce an *exactly zero* thickness. Moreover, in these works it is not reported any connections with a negative *J-integral*, nor comparisons with existing analytical solutions, whereas instead we make use of the relation in Chen (1986) and a numerically computed *J-integral* to validate our results in terms of stress intensity factors.

Exploiting the strong relation with cracks, we used an idea recently developed (Barbieri et al., 2012; Barbieri and Petrinic, 2013b,a) for fracture in a meshfree context: the aim is to create a crack where the RLI is positioned, and then impose a rigid motion at the (anti) cracks surfaces. The orientation of the inclusion can be arbitrary inside the matrix, without restrictions imposed by the underlying discretization of the matrix.

The structure of the paper is the following: Section 2 summarizes the analytical solutions available in the literature, alongside with the formulas for the extraction of the SIFs; Section 3 describes the governing equations in strong and weak form and the ones arising from their discretization; Section 4 presents the examples for the validation of the method, comparison with analytical solutions (full field and SIFs) and reorientation of fibers under a tensile loading; finally, in Section 5 conclusions are drawn.

2. Analytical solutions, J-integral and stress intensity factors

2.1. Analytical solution

Atkinson (1973) derived an analytical solution for an horizontal rigid line inclusion problem in an infinite isotropic elastic matrix under uniform remote biaxial loading σ_x^∞ and σ_y^∞ . In the following, the orthogonal reference has axis x aligned with the inclusion with the origin in its middle point. The rigid line inclusion has length $2a$.

Under uniform biaxial tension, and without the inclusion, the matrix strains uniformly, with a displacements field given by

$$u_0(x, y) = \frac{x}{8\mu} \left((\kappa + 1)\sigma_x^\infty + (\kappa - 3)\sigma_y^\infty \right) \quad (1)$$

$$v_0(x, y) = \frac{y}{8\mu} \left((\kappa - 3)\sigma_x^\infty + (\kappa + 1)\sigma_y^\infty \right) \quad (2)$$

where κ is

$$\kappa = \begin{cases} 3 - 4\nu & \text{plane strain} \\ \frac{3-\nu}{1+\nu} & \text{plane stress} \end{cases} \quad (3)$$

and μ is the *shear modulus* and ν is the Poisson ratio. The component ϵ_{x0} of the strain tensor given by

$$\epsilon_{x0}(x, y) = \frac{1}{8\mu} \left((\kappa + 1)\sigma_x^\infty + (\kappa - 3)\sigma_y^\infty \right) \quad (4)$$

The line inclusion can only move rigidly. Hence, with the rigid line inclusion now inserted in the matrix, and for the symmetry of the problem, the motion is only translational in the horizontal direction and with no rotation. For the compatibility of the displacements, this translation must be equal to the displacement u_0 (1) at its tips ($x = \pm a$, $y = 0$). In deriving the analytical solution, Atkinson, 1973 conveniently subtracted out the uniform strain of the matrix to obtain zero stresses at infinity. Hence,

$$\begin{cases} u(x, 0) = -u_0 & |x| < a \\ v(x, 0) = 0 & \forall x \end{cases} \quad (5)$$

and, for the equilibrium, $\tau_{xy} = 0$ $|x| > a$. Additionally, all stresses tend to infinity for x tending to zero. With these boundary conditions, the stress tensor at $y = 0$ (1) has components

$$\sigma_x = -(\kappa + 3)A \left(1 - \frac{|x|}{\sqrt{x^2 - a^2}}\right) \quad |x| > a \quad (6)$$

$$\sigma_y = (\kappa - 1)A \left(1 - \frac{|x|}{\sqrt{x^2 - a^2}}\right) \quad |x| > a \quad (7)$$

$$\tau_{xy} = \frac{\kappa + 1}{\sqrt{a^2 - x^2}} Ax \quad |x| < a \quad (8)$$

where A is

$$A = \frac{1}{8} \left((\kappa + 1)\sigma_x^\infty + (\kappa - 3)\sigma_y^\infty \right) \quad (9)$$

2.2. Asymptotic near-tip solution

Wang et al. (1985) derived a near-tip expansion of the stress fields that resembles the asymptotic crack-tip solution, namely

$$\begin{bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{bmatrix} = \frac{H_I}{\sqrt{2\pi r}} \begin{bmatrix} \cos \frac{\theta}{2} \left(\frac{\kappa+3}{2} - \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right) \\ \cos \frac{\theta}{2} \left(\frac{1-\kappa}{2} + \sin \frac{\theta}{2} \sin \frac{3\theta}{2} \right) \\ \sin \frac{\theta}{2} \left(\frac{1+\kappa}{2} + \cos \frac{\theta}{2} \cos \frac{3\theta}{2} \right) \end{bmatrix} \quad (10)$$

where r and θ is a polar coordinate system (Fig. 1) with $r \ll a$. The parameter H_I is not an equivalent stress intensity factor (SIF): however, it can be related to the remote loading conditions and to the value of the J -integral, as it will be shown next.

It easy to realize that the Mode I SIF K_I is then

$$K_I = \frac{1 - \kappa}{2} H_I \quad (11)$$

Moreover, according to Eq. (11), K_I and H_I might be of a different sign.

2.3. Relation between SIF and analytical solution

From Eq. (7) and assuming $r \ll a$, with $r = x - a$

$$\sigma_y \approx (1 - \kappa)A \left(\frac{r+a}{\sqrt{r^2 + a^2 + 2ra - a^2}} \right) \approx (1 - \kappa)A \left(\frac{a}{\sqrt{2ra}} \right) = \frac{(1 - \kappa)A\sqrt{a}}{\sqrt{2}} \frac{1}{\sqrt{r}} \quad (12)$$

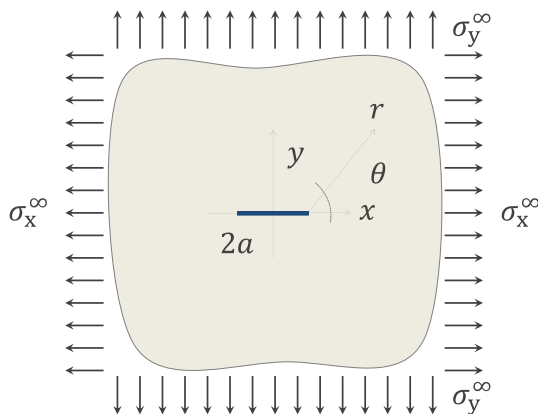


Fig. 1. Reference frame and loading for the analytical solution (Atkinson, 1973).

Comparing with the second of (10)

$$H_I = 2A\sqrt{\pi a} \quad (13)$$

and, for (11)

$$K_I = (1 - \kappa)A\sqrt{\pi a} \quad (14)$$

2.4. Relation between the SIF and the J -integral

As reported in Chen (1986), the J -integral for anticracks is negative. Using an integral domain method (Walters et al., 2005), the J -integral on line inclusions can be computed as follows

$$J = \int_A \boldsymbol{\sigma} \frac{\partial \mathbf{u}}{\partial X} \nabla q \, dA - \int_A W \frac{\partial q}{\partial X} \, dA - \int_{S^+ \cup S^-} \mathbf{t}^T \frac{\partial \mathbf{u}}{\partial X} \, dS \quad (15)$$

where A is a domain enclosing the crack tip. In Eq. (15) an additional term appears that takes into account the presence of non-zero tractions \mathbf{t} on the line inclusion, and $S^+ \cup S^-$ are the two sides of the anticrack. Furthermore, q is a virtual displacements scalar weight function, and W is the strain energy density. More details can be found in Appendix A.

Chen proved that

$$J = -\frac{\kappa(\kappa + 1)}{8\mu} H_I^2 \quad (16)$$

Hence, the numerical SIF can be extracted from Eq. (16) after having computed J from Eq. (16)

$$H_I^{num} = \sqrt{-J \frac{8\mu}{\kappa(\kappa + 1)}} \quad (17)$$

3. Governing equations

3.1. Strong form

Justified by the assumption of soft and deformable matrix, common in biological tissues, the kinematics in this paper is one of a finite deformation. In addition, finite deformation of the matrix allows to demonstrate the effectiveness of the numerical method for large rotations of the RLI. The continuum formulation is Total Lagrangian (TL), with Ω_0 being the reference configuration, and Ω the deformed configuration. In the following, \mathbf{X} denote the material coordinates and \mathbf{x} the deformation. We will also consider static problems, and for ease of readability, this section treats only one anticrack. However, the equations extend easily to multiple anticracks, as showed in Section 3.5.

In a discrete meshfree setting, the anticracks can be explicitly introduced by the intrinsic enrichment presented in Barbieri et al. (2012) and Barbieri and Petrinic (2013b,a), which is based only on geometry (distance fields) therefore completely independent from the particular constitutive model.

The Boundary Value Problem (BVP) is the following: find the displacement fields $\mathbf{u} : \Omega_0 \subset \mathbb{R}^3 \rightarrow \mathbb{R}^3$ such that it satisfies the following equilibrium equations in a Lagrangian description, in absence of body forces:

$$\begin{cases} \nabla_0 \cdot \mathbf{P} = 0 & \mathbf{X} \in \Omega_0 \\ \mathbf{n}_0 \cdot \mathbf{P} = \mathbf{t}_0 & \mathbf{X} \in \Gamma_t^0 \\ \mathbf{u} = \bar{\mathbf{u}} & \mathbf{X} \in \Gamma_u^0 \end{cases} \quad (18)$$

where \mathbf{P} is the First Piola–Kirchhoff stress, \mathbf{n}_0 is the normal unity vector (in the reference configuration) of the boundary Γ_t^0 where the traction \mathbf{t}_0 is prescribed, and Γ_u^0 is the boundary where the displacement $\bar{\mathbf{u}}$ is prescribed. The symbol ∇_0 denotes the gradient with respect to the material coordinates.

In addition, a rigid motion must be imposed on the surfaces of the inclusion: calling S^+ the top side of the inclusion and S^- the bottom side

$$\begin{cases} \mathbf{u}^+ - \mathbf{u}_L = \mathbf{0} & \mathbf{X} \in S^+ \\ \mathbf{u}^- - \mathbf{u}_L = \mathbf{0} & \mathbf{X} \in S^- \end{cases} \quad (19)$$

where \mathbf{u}_L is the rigid motion of the inclusion, which can be decomposed in a rotation and a translation. In a TL framework, the undeformed configuration \mathbf{X}_L can be mapped into the current configuration of the inclusion \mathbf{x}_L

$$\mathbf{x}_L = \mathbf{c} + \mathbf{R} \mathbf{X}_L \quad (20)$$

where \mathbf{c} is the deformation of the center of rotation (which may not belong to the RLI), \mathbf{R} is a rotation matrix (in 2D)

$$\mathbf{R} = \begin{bmatrix} \cos \theta & -\sin \theta \\ \sin \theta & \cos \theta \end{bmatrix} \quad (21)$$

Thus, the displacement \mathbf{u}_L of the inclusion in Eq. (19) is given by

$$\mathbf{u}_L = \mathbf{x}_L - \mathbf{X}_L = \mathbf{c} + (\mathbf{R} - \mathbf{I})\mathbf{X}_L \quad (22)$$

or, in an expanded form

$$\begin{cases} u_L = c_x + (\cos \theta - 1) X_L - \sin \theta Y_L \\ v_L = c_y + \sin \theta X_L + (\cos \theta - 1) Y_L \end{cases} \quad (23)$$

From Eq. (23) emerges that there are 3 additional unknowns (c_x, c_y, θ) for each rigid inclusion. Such motion is not imposed a priori, but is a consequence of the equilibrium.

3.2. Weak form

Using the displacement \mathbf{u} as a test function for Eqs. (18), the variational form can be written as

$$\begin{aligned} \int_{\Omega_0} \delta \mathbf{E} : \mathbf{S} d\Omega_0 - \int_{\Gamma_t^0} \delta \mathbf{u}^T \mathbf{t}_0 d\Gamma_t^0 + \alpha \int_{\Gamma_u^0} \delta(\mathbf{u} - \bar{\mathbf{u}})^T (\mathbf{u} - \bar{\mathbf{u}}) d\Gamma_u^0 \\ + \alpha \int_{S^+} \delta(\mathbf{u}^+ - \mathbf{u}_L)^T (\mathbf{u}^+ - \mathbf{u}_L) dS^+ \\ + \alpha \int_{S^-} \delta(\mathbf{u}^- - \mathbf{u}_L)^T (\mathbf{u}^- - \mathbf{u}_L) dS^- = 0 \end{aligned} \quad (24)$$

where \mathbf{E} is the Green–Lagrange strain, \mathbf{S} is the Second Piola–Kirchhoff stress, α is a penalty parameter (usually a large number of the same order of magnitude of the elastic properties) used to enforce the essential boundary conditions. The Green Lagrange strain is defined as

$$\mathbf{E} = \frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{I}) \quad (25)$$

with \mathbf{F} being the deformation gradient. The Second Piola–Kirchhoff stress is related to \mathbf{P} and to the Cauchy stress $\boldsymbol{\sigma}$ with the following relations:

$$\mathbf{S} = \mathbf{F}^{-1} \mathbf{P} \quad \boldsymbol{\sigma} = \frac{1}{J} \mathbf{F} \mathbf{S} \mathbf{F}^T \quad (26)$$

with J being the Jacobian of the deformation gradient. The variations in (24) can be expanded as follows:

$$\delta(\mathbf{u}^+ - \mathbf{u}_L)^T = \delta \mathbf{u}^{+T} - \delta \mathbf{u}_L^T \quad (27)$$

where

$$\delta \mathbf{u}^{+T} = \delta \mathbf{d}^T \begin{bmatrix} \phi_+ & 0 \\ 0 & \phi_+ \end{bmatrix} \quad (28)$$

where ϕ_+ are the shape functions of the discretization nodes on S^+ and \mathbf{d} the global vector of the nodal unknowns. From Eq. (18),

$$\delta \mathbf{u}_L^T = \delta \mathbf{c}^T + \delta \theta [-\sin \theta X_L - \cos \theta Y_L \quad \cos \theta X_L - \sin \theta Y_L] \quad (29)$$

Therefore, the penalty term of the rigid line inclusion in Eq. (24) for S^+ can be written as

$$\begin{aligned} \alpha \delta \mathbf{d}^T \int_{S^+} \begin{bmatrix} \phi_+(u^+ - u_L) \\ \phi_+(v^+ - v_L) \end{bmatrix} dS^+ + \alpha \delta \mathbf{c}^T \int_{S^+} \begin{bmatrix} (u_L - u^+) \\ (v_L - v^+) \end{bmatrix} dS^+ \\ + \alpha \delta \theta \int_{\Gamma_c} (\sin \theta X_L + \cos \theta Y_L)(u^+ - u_L) \\ + (-\cos \theta X_L + \sin \theta Y_L)(v^+ - v_L) dS^+ \end{aligned} \quad (30)$$

and similarly for S^- .

3.3. Discretized equations of motion

The approximation \mathbf{u}^h of the field variable $\mathbf{u}(\mathbf{X})$ in Eq. (18) is an expansion

$$\mathbf{u}^h(\mathbf{X}) = \sum_I^N \phi_I(\mathbf{X}) \mathbf{d}_I \quad (31)$$

where N is the number of discretization points, $\phi_I : \Omega \rightarrow \mathbb{R}$ is the I -th shape function and \mathbf{d}_I is the nodal value related to the position \mathbf{X}_I . The expressions for the shape functions in Eq. (31) depend on the method employed for the discretization: in this paper we use the Reproducing Kernel Particle Method (RKPM) (Liu et al., 1995) because it allows a fast introduction of crack surfaces without *remeshing* (Barbieri and Meo, 2012; Barbieri et al., 2012; Barbieri and Petrinic, 2013a,b). On these crack surfaces, a rigid motion (23) is enforced.

Following Belytschko et al. (2000), using Eq. (31), the discretized variation of the deformation gradient becomes

$$\delta \mathbf{F}^h = \delta \mathbf{d} \mathcal{B}_0^T \quad (32)$$

with

$$\mathcal{B}_0^T = \begin{bmatrix} \frac{\partial \phi_I^T}{\partial X} & \frac{\partial \phi_I^T}{\partial Y} & \frac{\partial \phi_I^T}{\partial Z} \end{bmatrix} \quad (33)$$

and the following holds

$$\delta \mathbf{F} : \mathbf{P} = \delta \mathbf{E} : \mathbf{S} \quad (34)$$

Replacing (31) into (18) and considering Eq. (30), the following equilibrium equations are obtained

$$\begin{cases} \mathbf{F}^{(i)}(\mathbf{d}) - \mathbf{F}^{(e)} + \mathbf{F}_d^+(\mathbf{d}, \mathbf{c}, \theta) + \mathbf{F}_d^-(\mathbf{d}, \mathbf{c}, \theta) = \mathbf{0} \\ \mathbf{F}_c^+(\mathbf{d}, \mathbf{c}, \theta) + \mathbf{F}_c^-(\mathbf{d}, \mathbf{c}, \theta) = \mathbf{0} \\ F_\theta^+(\mathbf{d}, \mathbf{c}, \theta) + F_\theta^-(\mathbf{d}, \mathbf{c}, \theta) = 0 \end{cases} \quad (35)$$

where $\mathbf{F}^{(e)}$ is the external forces vector

$$\mathbf{F}_I^e = \int_{\Gamma_u} \phi_I^T \mathbf{t}_0 d\Gamma_u \quad (36)$$

\mathbf{F}_d^+ (and similarly \mathbf{F}_d^-) is the coupling between the matrix and the inclusion

$$\mathbf{F}_d^+(\mathbf{d}, \mathbf{c}, \theta) = \alpha \int_{S^+} \begin{bmatrix} \phi_+(u^+ - u_L) \\ \phi_+(v^+ - v_L) \end{bmatrix} dS^+ \quad (37)$$

\mathbf{F}_c^+ (and similarly \mathbf{F}_c^-)

$$\mathbf{F}_c^+(\mathbf{d}, \mathbf{c}, \theta) = \alpha \int_{S^+} \begin{bmatrix} (u_L - u^+) \\ (v_L - v^+) \end{bmatrix} dS^+ \quad (38)$$

F_θ^+ (and similarly F_θ^-)

$$F_{\theta}^{+}(\mathbf{d}, \mathbf{c}, \theta) = \int_{S^{+}} (\sin \theta X_L + \cos \theta Y_L)(u^{+} - u_L) + (-\cos \theta X_L + \sin \theta Y_L)(v^{+} - v_L) dS^{+} \quad (39)$$

and finally, $\mathbf{F}^{(i)}$ is the *internal forces* vector that depends on the *constitutive model* of the material

$$\begin{aligned} \delta \mathbf{d}^T \mathbf{F}^{(i)}(\mathbf{d}) &= \int_{\Omega_0} \delta \mathbf{E}^h : \mathbf{S}^h d\Omega_0 = \int_{\Omega_0} \delta \mathbf{F}^h : \mathbf{P}^h d\Omega_0 \\ &= \delta \mathbf{d}^T \int_{\Omega_0} \begin{bmatrix} \frac{\partial \phi}{\partial X} P_{11}(\mathbf{d}) + \frac{\partial \phi}{\partial Y} P_{21}(\mathbf{d}) \\ \frac{\partial \phi}{\partial X} P_{12}(\mathbf{d}) + \frac{\partial \phi}{\partial Y} P_{22}(\mathbf{d}) \end{bmatrix} d\Omega_0 \end{aligned} \quad (40)$$

3.4. Tangent stiffness matrix

Eq. (35) is nonlinear in $(\mathbf{d}, \mathbf{c}, \theta)$. For n inclusions, the total number of unknowns is then $2N + 3n$ in two dimensions. Therefore, solving (35) requires an iterative numerical scheme (for instance *Newton–Raphson*). For each iteration, it is necessary to compute the *tangent stiffness matrix* (or the *Jacobian*) K_T that is the gradient of the left-hand side of Eq. (35) with respect to the unknowns $(\mathbf{d}, \mathbf{c}, \theta)$. This gradient has the following form:

$$\mathbf{K}_T = \begin{bmatrix} \frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} + \frac{\partial \mathbf{F}_d^+}{\partial \mathbf{d}} + \frac{\partial \mathbf{F}_d^-}{\partial \mathbf{d}} & \frac{\partial \mathbf{F}_d^+}{\partial \mathbf{c}} + \frac{\partial \mathbf{F}_d^-}{\partial \mathbf{c}} & \frac{\partial \mathbf{F}_d^+}{\partial \theta} + \frac{\partial \mathbf{F}_d^-}{\partial \theta} \\ \frac{\partial \mathbf{F}_c^+}{\partial \mathbf{d}} + \frac{\partial \mathbf{F}_c^-}{\partial \mathbf{d}} & \frac{\partial \mathbf{F}_c^+}{\partial \mathbf{c}} + \frac{\partial \mathbf{F}_c^-}{\partial \mathbf{c}} & \frac{\partial \mathbf{F}_c^+}{\partial \theta} + \frac{\partial \mathbf{F}_c^-}{\partial \theta} \\ \frac{\partial \mathbf{F}_{\theta}^+}{\partial \mathbf{d}} + \frac{\partial \mathbf{F}_{\theta}^-}{\partial \mathbf{d}} & \frac{\partial \mathbf{F}_{\theta}^+}{\partial \mathbf{c}} + \frac{\partial \mathbf{F}_{\theta}^-}{\partial \mathbf{c}} & \frac{\partial \mathbf{F}_{\theta}^+}{\partial \theta} + \frac{\partial \mathbf{F}_{\theta}^-}{\partial \theta} \end{bmatrix} \quad (41)$$

We will show in the following that the matrix in (41) is symmetric. Indeed, the entries are

$$\frac{\partial \mathbf{F}_d^+}{\partial \mathbf{d}} = \int_{S^+} \begin{bmatrix} \phi_+ \phi_+^T & 0 \\ 0 & \phi_+ \phi_+^T \end{bmatrix} dS^+ \quad (42)$$

$$\frac{\partial \mathbf{F}_d^+}{\partial \mathbf{c}} = \int_{S^+} \begin{bmatrix} -\phi_+ & 0 \\ 0 & -\phi_+ \end{bmatrix} dS^+ = \frac{\partial \mathbf{F}_c^+}{\partial \mathbf{d}} \quad (43)$$

$$\frac{\partial \mathbf{F}_d^+}{\partial \theta} = \int_{S^+} \begin{bmatrix} \phi_+ (\sin \theta X_L + \cos \theta Y_L) \\ \phi_+ (-\cos \theta X_L + \sin \theta Y_L) \end{bmatrix} dS^+ = \frac{\partial \mathbf{F}_{\theta}^+}{\partial \mathbf{d}} \quad (44)$$

$$\frac{\partial \mathbf{F}_c^+}{\partial \mathbf{c}} = \int_{S^+} dS^+ \mathbf{I} \quad (45)$$

where \mathbf{I} is the *identity* matrix,

$$\frac{\partial \mathbf{F}_c^+}{\partial \theta} = \int_{S^+} \begin{bmatrix} -(\sin \theta X_L + \cos \theta Y_L) \\ -(-\cos \theta X_L + \sin \theta Y_L) \end{bmatrix} dS^+ = \frac{\partial \mathbf{F}_{\theta}^+}{\partial \mathbf{c}} \quad (46)$$

$$\frac{\partial \mathbf{F}_{\theta}^+}{\partial \theta} = \int_{S^+} (\sin \theta X_L + \cos \theta Y_L)^2 + (\cos \theta X_L - \sin \theta Y_L)^2 dS^+ \quad (47)$$

It can be shown that

$$\frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} = \left(\frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} \right)_g + \left(\frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} \right)_m \quad (48)$$

where subscript $(\cdot)_g$ stands for *geometric part*, which takes into account geometrical nonlinearities

$$\left(\frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} \right)_g = \int_{\Omega_0} \begin{bmatrix} \mathcal{B}_0^T \mathbf{S}(\mathbf{d}) \mathcal{B}_0 \\ \mathcal{B}_0^T \mathbf{S}(\mathbf{d}) \mathcal{B}_0 \end{bmatrix} d\Omega_0 \quad (49)$$

whilst $(\cdot)_m$ stands for the *material part*, which takes into account material nonlinearities

$$\left(\frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} \right)_m = \int_{\Omega_0} \mathbf{B}^T(\mathbf{d}) \mathbf{C}^{SE}(\mathbf{d}) \mathbf{B}(\mathbf{d}) d\Omega_0 \quad (50)$$

where

$$\mathbf{B}(\mathbf{d}) = \begin{bmatrix} F_{11} \frac{\partial \phi^T}{\partial X} & F_{21} \frac{\partial \phi^T}{\partial X} \\ F_{12} \frac{\partial \phi^T}{\partial Y} & F_{22} \frac{\partial \phi^T}{\partial Y} \\ F_{11} \frac{\partial \phi^T}{\partial Y} + F_{12} \frac{\partial \phi^T}{\partial X} & F_{21} \frac{\partial \phi^T}{\partial Y} + F_{22} \frac{\partial \phi^T}{\partial X} \end{bmatrix} \quad (51)$$

and \mathbf{C}^{SE} is the *Second Elasticity Tensor* in Voigt form, defined, in tensorial form, as

$$\mathbb{C}^{SE} = \frac{\partial \mathbf{S}}{\partial \mathbf{E}} \quad (52)$$

3.5. Multiple rigid line inclusions

For multiple anticracks, Eq. (24) modifies as

$$\begin{aligned} \int_{\Omega_0} \delta \mathbf{E} : \mathbf{S} d\Omega_0 - \int_{\Gamma_t^0} \delta \mathbf{u}^T \mathbf{t}_0 d\Gamma_t^0 + \alpha \int_{\Gamma_u^0} \delta(\mathbf{u} - \bar{\mathbf{u}})^T (\mathbf{u} - \bar{\mathbf{u}}) d\Gamma_u^0 \\ + \sum_{i=1}^{n_a} \alpha \int_{S_i^+} \delta(\mathbf{u}^+ - \mathbf{u}_{L,i})^T (\mathbf{u}^+ - \mathbf{u}_{L,i}) dS_i^+ \\ + \alpha \int_{S_i^-} \delta(\mathbf{u}^- - \mathbf{u}_{L,i})^T (\mathbf{u}^- - \mathbf{u}_{L,i}) dS_i^- = 0 \end{aligned} \quad (53)$$

with n_a being the number of anticracks. With similar steps as in Sections 3.1 and 3.2, the discretized equations of motion are

$$\begin{cases} \mathbf{F}^{(i)}(\mathbf{d}) - \mathbf{F}^{(e)} + \sum_{i=1}^{n_a} \mathbf{F}_d^+(\mathbf{d}, \mathbf{c}_i, \theta_i) + \mathbf{F}_d^-(\mathbf{d}, \mathbf{c}_i, \theta_i) = 0 \\ \mathbf{F}_{c_1}^+(\mathbf{d}, \mathbf{c}_1, \theta_1) + \mathbf{F}_{c_1}^-(\mathbf{d}, \mathbf{c}_1, \theta_1) = 0 \\ \mathbf{F}_{c_2}^+(\mathbf{d}, \mathbf{c}_2, \theta_2) + \mathbf{F}_{c_2}^-(\mathbf{d}, \mathbf{c}_2, \theta_2) = 0 \\ \dots \\ \mathbf{F}_{c_{n_a}}^+(\mathbf{d}, \mathbf{c}_{n_a}, \theta_{n_a}) + \mathbf{F}_{c_{n_a}}^-(\mathbf{d}, \mathbf{c}_{n_a}, \theta_{n_a}) = 0 \\ F_{\theta_1}^+(\mathbf{d}, \mathbf{c}_1, \theta_1) + F_{\theta_1}^-(\mathbf{d}, \mathbf{c}_1, \theta_1) = 0 \\ F_{\theta_2}^+(\mathbf{d}, \mathbf{c}_2, \theta_2) + F_{\theta_2}^-(\mathbf{d}, \mathbf{c}_2, \theta_2) = 0 \\ \dots \\ F_{\theta_{n_a}}^+(\mathbf{d}, \mathbf{c}_{n_a}, \theta_{n_a}) + F_{\theta_{n_a}}^-(\mathbf{d}, \mathbf{c}_{n_a}, \theta_{n_a}) = 0 \end{cases} \quad (54)$$

where the terms are analogous to Eqs. (37)–(39). Similarly, the tangent stiffness matrix is given by:

$$\mathbf{K}_T = \begin{bmatrix} \frac{\partial \mathbf{F}^{(i)}}{\partial \mathbf{d}} + \sum_{i=1}^{n_a} \frac{\partial \mathbf{F}_d^+}{\partial \mathbf{d}} + \frac{\partial \mathbf{F}_d^-}{\partial \mathbf{d}} & \mathbf{K}_{dc}^T & \mathbf{K}_{d\theta}^T \\ \mathbf{K}_{dc} & \mathbf{K}_{cc} & \mathbf{K}_{c\theta}^T \\ \mathbf{K}_{d\theta} & \mathbf{K}_{c\theta} & \mathbf{K}_{\theta\theta} \end{bmatrix} \quad (55)$$

where

$$\mathbf{K}_{dc}^T = \left[\frac{\partial \mathbf{F}_d^+}{\partial \mathbf{c}_1} + \frac{\partial \mathbf{F}_d^-}{\partial \mathbf{c}_1} \quad \frac{\partial \mathbf{F}_d^+}{\partial \mathbf{c}_2} + \frac{\partial \mathbf{F}_d^-}{\partial \mathbf{c}_2} \quad \dots \quad \frac{\partial \mathbf{F}_d^+}{\partial \mathbf{c}_{n_a}} + \frac{\partial \mathbf{F}_d^-}{\partial \mathbf{c}_{n_a}} \right] \quad (56)$$

$$\mathbf{K}_{d\theta}^T = \left[\frac{\partial \mathbf{F}_d^+}{\partial \theta_1} + \frac{\partial \mathbf{F}_d^-}{\partial \theta_1} \quad \frac{\partial \mathbf{F}_d^+}{\partial \theta_2} + \frac{\partial \mathbf{F}_d^-}{\partial \theta_2} \quad \dots \quad \frac{\partial \mathbf{F}_d^+}{\partial \theta_{n_a}} + \frac{\partial \mathbf{F}_d^-}{\partial \theta_{n_a}} \right] \quad (57)$$

$$\mathbf{K}_{c\theta} = \begin{bmatrix} \frac{\partial \mathbf{F}_{c_1}^+}{\partial \theta_1} + \frac{\partial \mathbf{F}_{c_1}^-}{\partial \theta_1} & \dots \\ \dots & \frac{\partial \mathbf{F}_{c_2}^+}{\partial \theta_2} + \frac{\partial \mathbf{F}_{c_2}^-}{\partial \theta_2} & \dots \\ \dots & \dots & \dots \\ \dots & \dots & \dots & \frac{\partial \mathbf{F}_{c_{n_a}}^+}{\partial \theta_{n_a}} + \frac{\partial \mathbf{F}_{c_{n_a}}^-}{\partial \theta_{n_a}} \end{bmatrix} \quad (58)$$

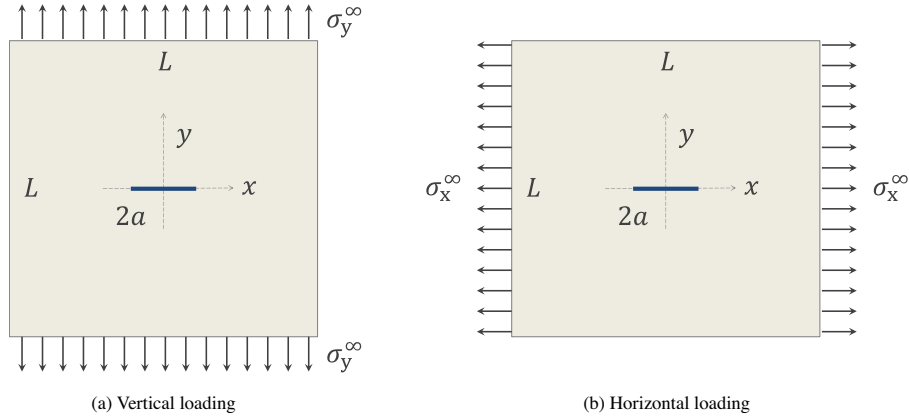


Fig. 2. Mode I anticrack. Thin black line: rigid line inclusion.

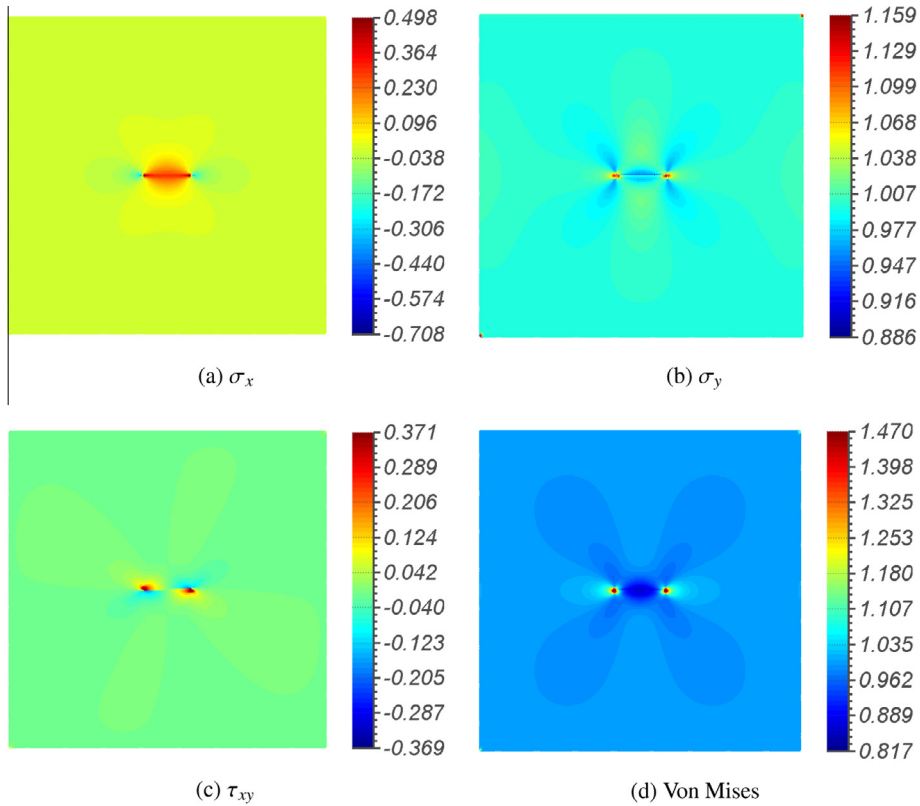


Fig. 3. Mode I under vertical loading: stress tensor normalized to σ_y^∞ , $L = 1$ m and $a = 0.075 L$.

$$\mathbf{K}_{cc} = \begin{bmatrix} \frac{\partial F_{c_1}^+}{\partial c_1} + \frac{\partial F_{c_1}^-}{\partial c_1} & \dots & \dots \\ \frac{\partial F_{c_2}^+}{\partial c_2} + \frac{\partial F_{c_2}^-}{\partial c_2} & \dots & \dots \\ \dots & \dots & \dots \\ \dots & \dots & \frac{\partial F_{c_{na}}^+}{\partial c_{na}} + \frac{\partial F_{c_{na}}^-}{\partial c_{na}} \end{bmatrix} \quad (59)$$

$$\mathbf{K}_{\theta\theta} = \begin{bmatrix} \frac{\partial F_{\theta_1}^+}{\partial \theta_1} + \frac{\partial F_{\theta_1}^-}{\partial \theta_1} & \dots & \dots \\ \frac{\partial F_{\theta_2}^+}{\partial \theta_2} + \frac{\partial F_{\theta_2}^-}{\partial \theta_2} & \dots & \dots \\ \dots & \dots & \dots \\ \dots & \dots & \frac{\partial F_{\theta_{na}}^+}{\partial \theta_{na}} + \frac{\partial F_{\theta_{na}}^-}{\partial \theta_{na}} \end{bmatrix} \quad (60)$$

where the entries in these matrices are analogous to Eqs. (42)–(47).

4. Numerical examples

4.1. Comparisons of SIFs with analytical solutions

For ease of comparison with the analytical solutions (6)–(8) and (10), the finite deformation in this section is linearized to reconcile with the theory described in Section 2. The matrix is an isotropic linear elastic body, defined by shear modulus μ and Poisson ratio ν , both in plane stress and plane strain. Nonetheless the approach is general, and applicable to non-linear constitutive models, as in Section 4.7. The nodal spacing (or *mesh size*) will be indicated in the following with h . The radius for the numerical computation of the *J-integral* (see Appendix A) in all cases was varied from $0.4 a$ to $0.7 a$ to verify the path independence of the *J-integral*. The values are $0.5 a$ are reported in the following tables.

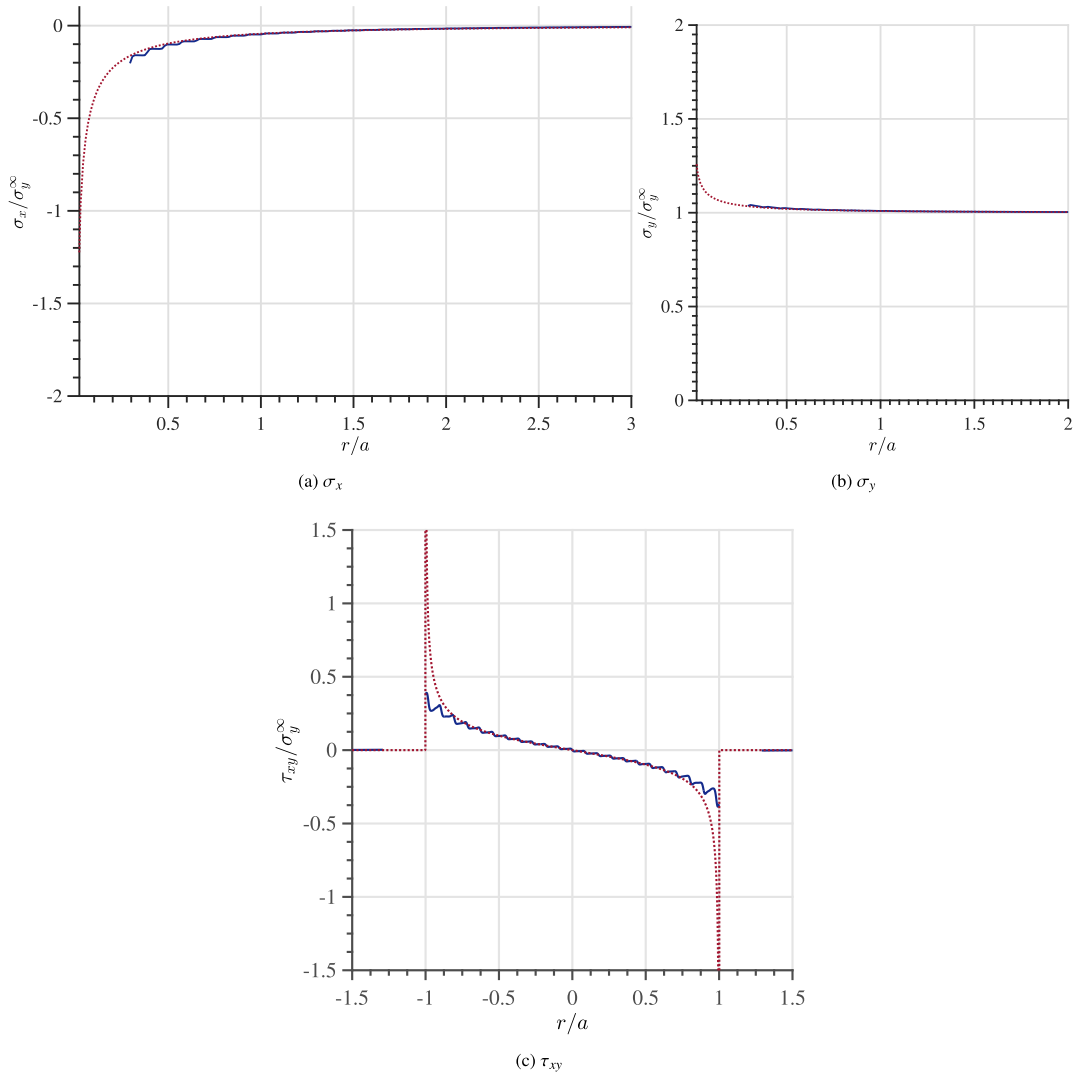


Fig. 4. Numerical (continuous line) and analytical (dashed line) solutions (Atkinson, 1973) according to Eqs. (6)–(8) for the vertical tensile loading (Fig. 2a).

4.2. Mode I under vertical tensile loading

In this section we will consider a central horizontal line inclusion embedded in a rectangular matrix subjected to a vertical tensile loading (Fig. 2a). Fig. 3 shows the distribution of the stress tensor and the Von Mises stress inside the matrix: the stress concentrations at the tips of a relatively short inclusion (0.15 L) are clearly visible. These concentrations are singular for $r = 0$, as in Eqs. (8), (6) and (7). The functional space of the approximation (31) does not contain singular functions, the singularity cannot appear in the computed stresses: however, as visible in Fig. 4, the numerical solution can capture with great accuracy the analytical solution near the tips. Most importantly, the method does capture stress intensities. Indeed, a significant comparison is reported in Tables 1–3 where the numerical values of the

J-integrals (Eqs. (16) and (17)) for different mesh sizes are compared to the value H_I (Eq. (13)), which is connected to the theoretical Stress Intensity Factor K_I (Eq. (14)). Tables 1–3 show rapid convergence to the analytical value of H_I , for a relatively short inclusions (close to the hypothesis of infinite matrix, see Section 2.1). Table 3 shows that even with a small number of nodes per half-length (5) it is possible to obtain the analytical answer

Table 2
Convergence analysis for the test in Fig. 2a in plane strain with $a = 0.075 L$: $H_I = -0.0809 \text{ Pa}\sqrt{\text{m}}$.

h/a	a/h	H_I numerical [$\text{Pa}\sqrt{\text{m}}$]	Error %
0.133	7.50	0.0856	5.81
0.089	11.25	0.0846	4.57
0.067	15.00	0.0840	3.83

Table 1
Convergence analysis for the test in Fig. 2a in plane strain with $a = 0.01 L$: $H_I = 0.0934 \text{ Pa}\sqrt{\text{m}}$.

h/a	a/h	H_I numerical [$\text{Pa}\sqrt{\text{m}}$]	Error %
0.100	10	0.0960	2.78
0.067	15	0.0950	1.71
0.050	20	0.0945	1.18

Table 3
Convergence analysis for the test in Fig. 2a in plane strain with $a = 5\% L$: $H_I = -0.0661 \text{ Pa}\sqrt{\text{m}}$.

h/a	a/h	H_I numerical [$\text{Pa}\sqrt{\text{m}}$]	Error %
0.200	5.00	0.0696	5.30
0.133	7.50	0.0678	2.57
0.100	10.0	0.0658	0.46

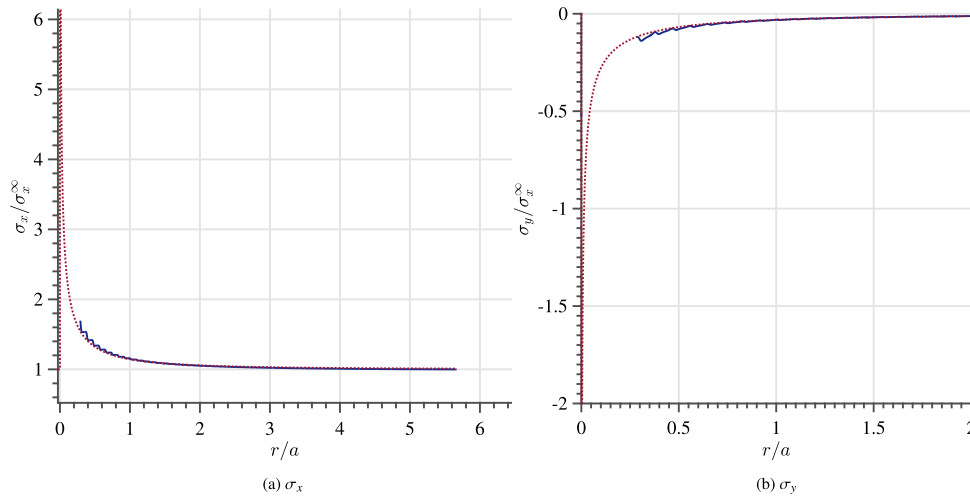


Fig. 5. Numerical (continuous line) and analytical (dashed line) solutions (Atkinson, 1973) according to Eqs. (6) and (7) for the horizontal tensile loading (Fig. 2b).

with a small error (around 5%). It is interesting to notice that for the cases in Tables 2 and 3 the values of H_I are negative. However, this does not mean a reversed stress singularity: using $\nu = 0.3$ and Young modulus $E = 50$ GPa, and applying Eq. (11), it is obtained a $K_I = 0.0324 \text{ Pa}\sqrt{\text{m}}$ for $a = 0.075 L$ and $K_I = 0.0264 \text{ Pa}\sqrt{\text{m}}$ for $a = 0.05 L$. The sign of K_I determines the sign of the singularity (see Eq. (10)), and it is found consistently to be positive.

4.3. Mode I under vertical compressive loading

Nonetheless, conversely to cracks, rigid line inclusion can have negative Mode I stress intensity factors, because the rigid motion of the inclusion always prevents a negative displacement jump. This case is the one in Section 4.2 ($a = 0.05 L$) with a reversed load, for which the computed stress intensity factor is $K_I = -0.0264 \text{ Pa}\sqrt{\text{m}}$. In this test, 5 nodes per half length were employed.

4.4. Mode I under horizontal tensile loading

Another difference from cracks, is the presence of a stress singularity for loading parallel to the inclusion. The value of K_I needs to be computed with a different formula than (14): indeed, applying (14) with $a = 0.075 L$, $\nu = 0.3$, $E = 50$ GPa and $\sigma_x^\infty > 0$, $\sigma_y^\infty = 0$ leads to $H_I = 0.1888 \text{ Pa}\sqrt{\text{m}}$ and $K_I = -0.0755 \text{ Pa}\sqrt{\text{m}}$, which is negative and counter-intuitive. In fact, Fig. 5a shows a positive singularity in the σ_x plot. The negative value so calculated corresponds to the σ_y distribution (Fig. 5b). To get a consistent K_I for Fig. 5a, let us consider Eq. (10): the expression for σ_x for $\theta = 0$ is

$$\sigma_y(r, \theta = 0) = \frac{H_I}{\sqrt{2\pi r}} \left(\frac{\kappa + 3}{2} \right) \quad (61)$$

hence, the correct expression for K_I under loading parallel to the anticrack is

$$K_I = (\kappa + 3)A\sqrt{\pi a} \quad (62)$$

Substituting the values in Eq. (62) leads to $K_I = 0.4530 \text{ Pa}\sqrt{\text{m}}$, which is correctly positive. Comparing the H_I values for different loading conditions is therefore more convenient than comparing K_I , because the expression for H_I is independent from the relative position between loading and inclusion.

Tables 4–6 show that good convergence is obtained also for the inclusion under parallel loading.

4.5. Mode I under horizontal compressive loading

Analogously to the vertical compressive loading in Section 4.3, there is a singularity in the stress also for horizontal compressive loading, as showed in Fig. 6, captured quite accurately by the proposed numerical scheme.

4.6. Mode I under biaxial loading

From the previous section, it emerged that an anticrack creates a singularity in the stress under any arbitrary biaxial loading, both tensile and compressive. However, this is true except in some circumstances. For instance, in plane strain for incompressible materials (Eq. (14)), for which $\kappa = 1$. For a compressible solid, K_I is null if $H_I = 0$, and, for Eq. (13), when $A = 0$. Hence, for Eq. (9), $A = 0$ for the following ratio between the loadings

$$\psi = \frac{\sigma_y^\infty}{\sigma_x^\infty} = -\frac{\kappa + 1}{\kappa - 3} \quad (63)$$

Table 4

Convergence analysis for the test in Fig. 2b in plane strain with $a = 0.10 L$: $H_I = 0.2180 \text{ Pa}\sqrt{\text{m}}$.

h/a	a/h	H_I numerical [$\text{Pa}\sqrt{\text{m}}$]	Error %
0.100	10	0.2221	1.88
0.067	15	0.2215	1.61
0.050	20	0.2200	0.92

Table 5

Convergence analysis for the test in Fig. 2b in plane strain with $a = 0.075 L$: $H_I = 0.1888 \text{ Pa}\sqrt{\text{m}}$.

h/a	a/h	H_I numerical [$\text{Pa}\sqrt{\text{m}}$]	Error %
0.133	7.50	0.1974	4.56
0.089	11.25	0.1958	3.71
0.067	15.00	0.1947	3.13

Table 6

Convergence analysis for the test in Fig. 2b in plane strain with $a = 0.05 L$: $H_I = 0.1577 \text{ Pa}\sqrt{\text{m}}$.

h/a	a/h	H_I numerical [$\text{Pa}\sqrt{\text{m}}$]	Error %
0.200	5.00	0.1618	5.00
0.133	7.50	0.1583	2.73
0.100	10.0	0.1577	2.34

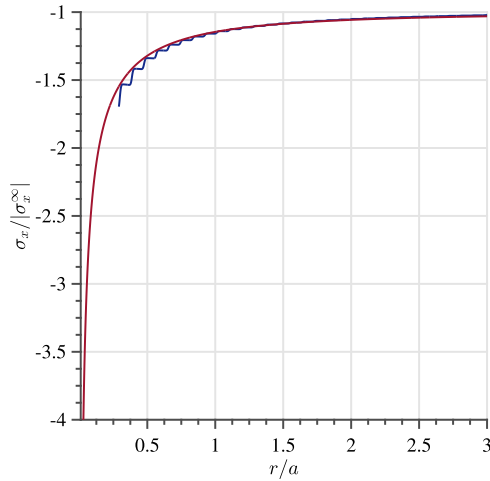


Fig. 6. Numerical (continuous line) and analytical (dashed line) solutions (Atkinson, 1973) according to Eq. (6) for horizontal compressive loading.

Fig. 7 show that for this ratio there is no singularity, and that the inclusion is *transparent* to the applied load: the stress state in the matrix is uniform.

4.7. Inclined multiple rigid line inclusions in a soft matrix

To test the numerical method for large rotations of the RLI, the following examples assume an initially inclined anticrack within a very deformable matrix. The domain is stretched uni-axially. The matrix follows a compressible hyperelastic Neo-Hookean model. The choice of a compressible model stands in avoiding numerical issues related to incompressibility. There is quite an abundant literature on the treatment of the incompressibility constraint in meshfree methods. However, the incompressibility issue is outside the scopes of the paper. This section wants to show that, under the assumption of finite deformation, the fibers rotate (and translate) in an evident manner. These results prove that the method can capture the physical intuition. For example, it can reproduce the reorientation of inclined fibers towards the direction of loading, phenomenon important in many biological systems (Tower et al., 2002). The alignment of the fibers with the loading has structural importance since, in this manner, reinforcements can be the most effective in withstanding loads. The reorientation is facilitated if the matrix is compliant. The strain energy function W of the hyperelastic model is

$$W = \frac{1}{2} \mu_0 (I_1 - 3) + \lambda_0 \frac{1}{2} (\log J)^2 - \mu_0 \log J \quad (64)$$

where μ_0 is the initial shear modulus, λ_0 the initial bulk modulus, I_1 is the first invariant of the Right Cauchy Green tensor

$$\mathbf{C} = \mathbf{F}^T \mathbf{F} \quad (65)$$

The corresponding Second Piola–Kirchhoff Stress is

$$\mathbf{S} = \mu_0 \mathbf{I} - (\mu_0 - \lambda_0 \log J) \mathbf{C}^{-1} \quad (66)$$

The second elasticity tensor (in index notation) is

$$(\mathbb{C}_{SE})_{ijkl} = \lambda_0 C_{ij}^{-1} C_{kl}^{-1} + (\mu_0 - \lambda_0 \log J) (C_{ik}^{-1} C_{jl}^{-1} + C_{il}^{-1} C_{kj}^{-1}) \quad (67)$$

In the following examples, unless differently specified, $\mu_0 = 0.4$ MPa, with initial Poisson ratio of $\nu = 0.3$. The domain of study is the same as the previous sections (unit square of $L = 1$ m) under uniaxial stretch λ in the Y direction. The midpoint of the RLI is in the middle of the square (see Fig. 8).

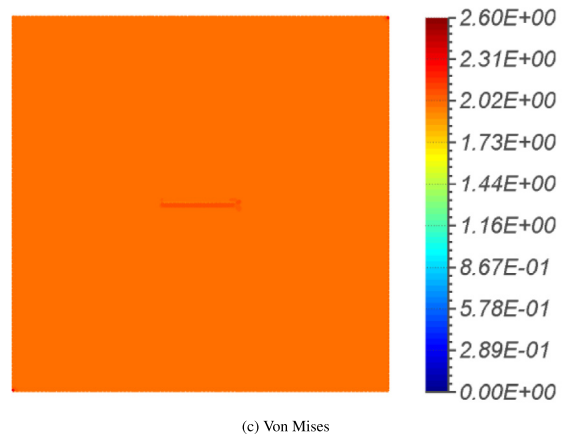
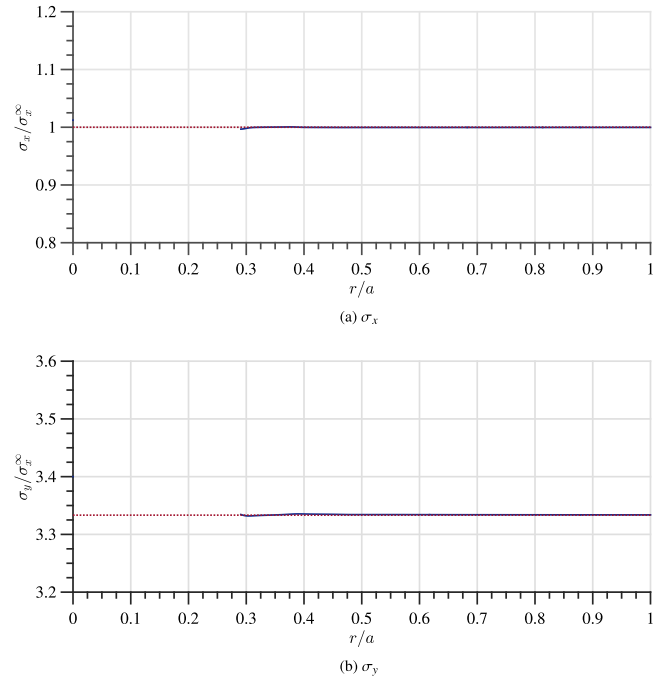


Fig. 7. Biaxial loading with ratio ψ in Eq. (63); numerical (continuous line) and analytical (dashed line) solutions (Atkinson, 1973). Note that the singularity disappears, as evidenced by the horizontal lines (Fig. 7a and b) and by the uniform distribution (Fig. 7c).

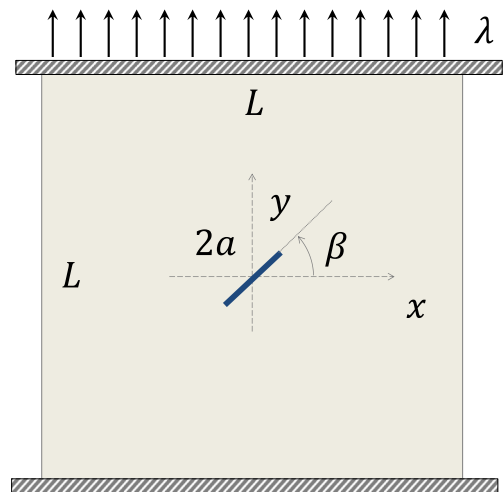


Fig. 8. Anticrack in mixed-mode loading under a large uniaxial stretch.

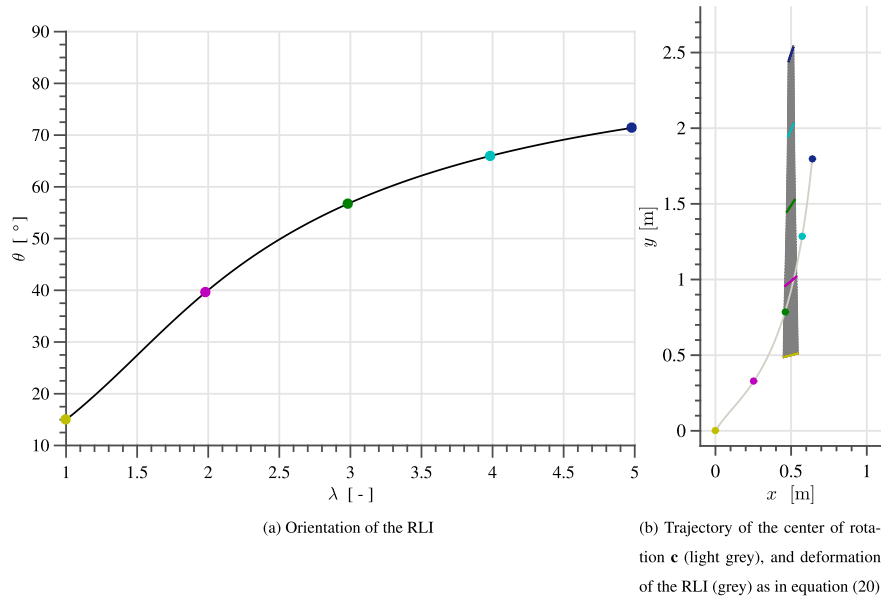


Fig. 9. Reorientation of a RLI with $\beta = 15^\circ$ embedded in a soft matrix, under uniaxial stretch: dots are color-coded with a fixed value of λ .

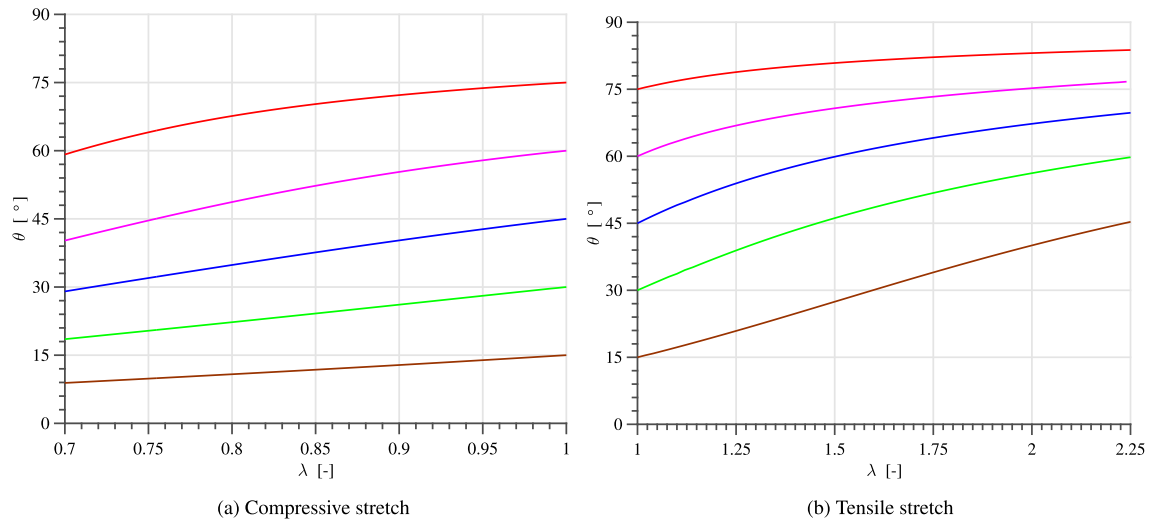


Fig. 10. Reorientation of a RLI for different β ; red: $\beta = 75^\circ$; magenta: $\beta = 60^\circ$; blue: $\beta = 45^\circ$; green: $\beta = 30^\circ$; brown: $\beta = 15^\circ$. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

4.7.1. One inclined anticrack: influence of large strains

The first series of results show the reorientation of the RLI under large values of λ . The length of the anticrack $2a$ is set to $L/10$, inclined with an angle β . The sample is discretized with a regular grid of 100×100 nodes, where the top and the bottom edges are clamped, and the top one free to move in the Y direction. The test is under *displacements-controlled* conditions, applied with an interval $\Delta\lambda = 0.02$. Figures in block 9 show the angle θ as a (non-linear) function of λ , and all the consequent deformations (translation and rotation) of a RLI with a slight initial inclination ($\beta = 15^\circ$). Under a large finite strain of 400%, the RLI rotates of 55° . At $\lambda = 5$ the simulation halted for excessive localized deformation near the RLI, which brought J to be negative. As expected, the midpoint of the RLI (which is different from the center of rotation c) translates only vertically.

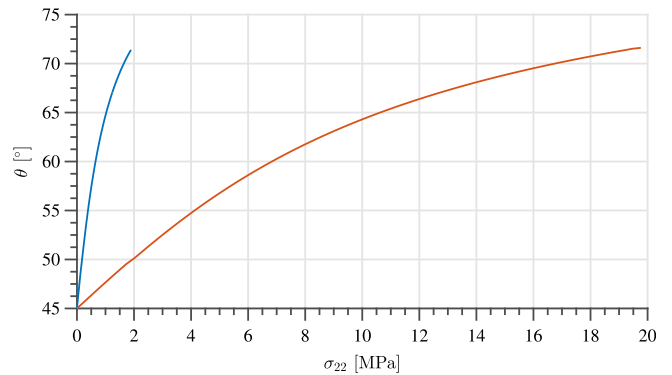


Fig. 11. Influence of the initial stiffness (with the same initial ν); blue: $\mu_0 = 0.42$ MPa; red: $\mu_0 = 4.2$ MPa. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

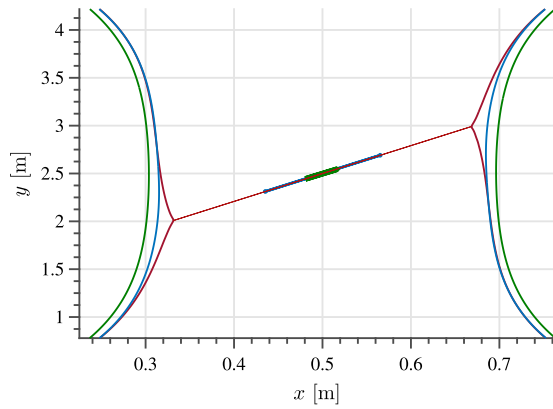


Fig. 12. Influence of the initial length a , showing a detail of the final position of the anticrack (inclined straight lines) and the final deformation; green: $2a = L/10$; blue: $2a = 2/5L$; red: RLi along the width. Axes are rescaled for enhanced visualization. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

4.7.2. One inclined anticrack: influence of the initial angle

Figures in block Fig. 10 show the reorientation of RLIs with different initial inclination. As expected, under tension, the reorientation is less for RLIs with high initial inclination, while the reverse verifies for compressive loadings. In fact, for compressive strains of 30%, the anticrack at $\beta = 75^\circ, 60^\circ, 45^\circ, 30^\circ$ reorientate of nearly

-15° . Some with a nonlinear variation ($\beta = 75^\circ, 60^\circ$), whilst the remaining with almost a linear law. Instead, the anticrack with $\beta = 15^\circ$ reorientates of -6° .

4.7.3. One inclined anticrack: influence of the initial stiffness

Fig. 11 shows the influence of the stiffness of the matrix on the reorientation of the anticrack, as obtained from the numerical results. As intuitively expected, the same load can reorientate the RLI of a larger amount than a stiffer matrix: for example, for a load of 2 MPa, a 10 times stiffer matrix reorientates of 5° , instead of more than 20° . From a different point of view, a stiffer matrix requires a higher stress than a more compliant matrix to reorientate of the same quantity. A 10 times stiffer matrix requires a 10 times higher stress. This is justifiable also from a theoretical perspective, where under uniaxial stress, the Cauchy stress along the direction of loading is given by

$$\sigma_{22} = \frac{\mu_0}{J^{5/3}} \left(\lambda^2 - \frac{J}{\lambda} \right) \tag{68}$$

where J is obtained by solving the following nonlinear equation

$$\lambda_0 \left(J^{8/3} - J^{5/3} \right) + \mu_0 \left(\frac{J}{3\lambda} - \frac{\lambda^2}{3} \right) = 0 \tag{69}$$

Eqs. (68) and (69) are valid in absence of anticracks, or, for relatively small ones, far from them. It is evident from Eq. (69) that under

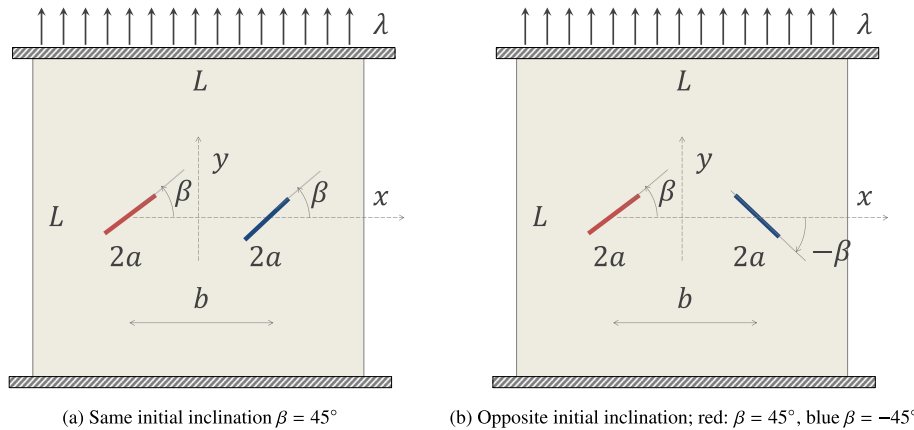


Fig. 13. Interaction of two anticracks: test cases and boundary conditions. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

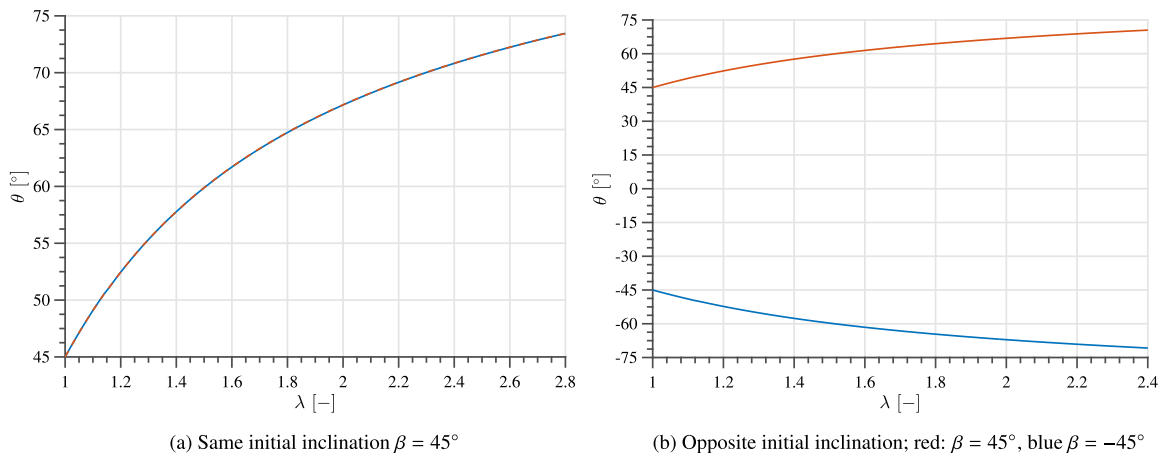


Fig. 14. Interaction of two anticracks: numerical results for $b = L/3$. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

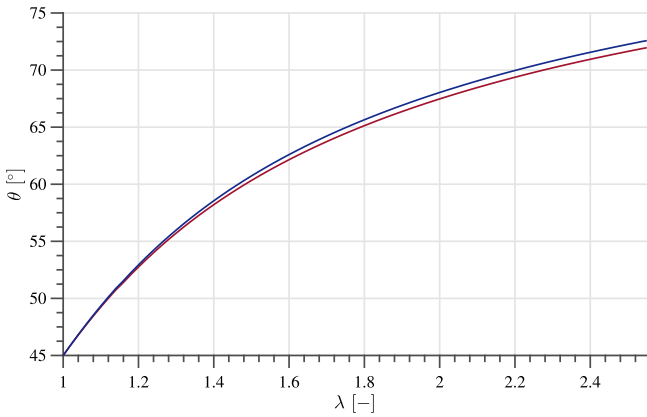


Fig. 15. Interaction of two anticracks: numerical results for same initial inclination $\beta = 45^\circ$ and $b = a$.

uniaxial stress, different μ_0 (and hence λ_0) give the same J . Therefore, for Eq. (68), σ_{22} is linearly proportional to the shear modulus.

4.7.4. One inclined anticrack: influence of the initial length

Fig. 12 shows the effect of the length of the anticrack on the deformation of the matrix. The initial angle is $\beta = 15^\circ$, with three different lengths, from relatively small, to the extreme case of an anticrack along the entire width. Fig. 12 is a zoom of the position

of the anticrack and the deformation of the edges for $\lambda = 5$. There is no influence of the length on the final rotation and position of the anticrack. However, as the length augments, the RLI increasingly strains the matrix. In the extreme case, the edges deform by creating a kink.

4.7.5. Two inclined anticracks

This section examines the mutual influence of the inclined anticracks as depicted in Fig. 13. The centers of the two RLIs are separated by a distance $b = L/3$, with $2a = L/10$. Fig. 14 shows the results of two different configurations: one with anticracks with equal inclination, and one with opposite. The colors of the curves are the same of the RLIs in Fig. 14. In both cases, for this value of b there is no influence on the rotation, and the two line inclusions deform as they were isolated, as reported in Fig. 10b (blue curve). To seek evidence for an interaction of the RLIs, the offset length b was reduced to half the length of the anticrack (a). Fig. 15 displays only a slight difference in θ compared to Fig. 14a, and at large strains. The influence is the same also for RLIs inclined in an opposite direction (not reported). For anticracks, the local stress state near the tips does not seem to hinder significantly their reorientation.

4.7.6. Multiple RLIs

The last example considers a large number of randomly oriented anticracks. This case proves the power of the method, especially in handling multiple couplings between the matrix and the

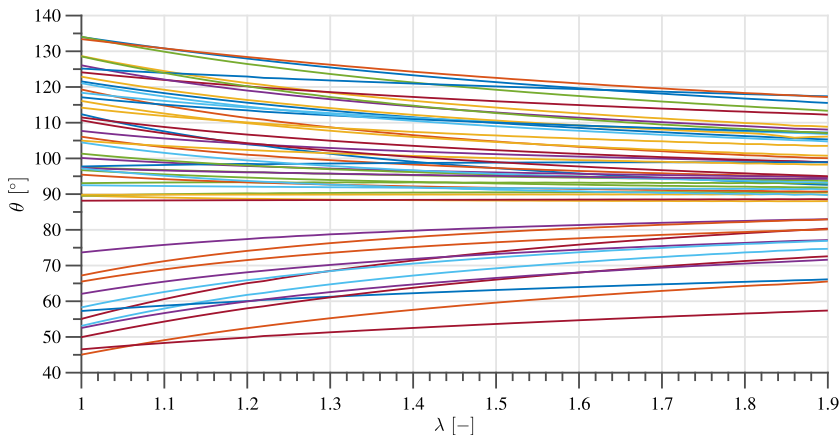


Fig. 16. Reorientation of 49 randomly oriented anticracks.

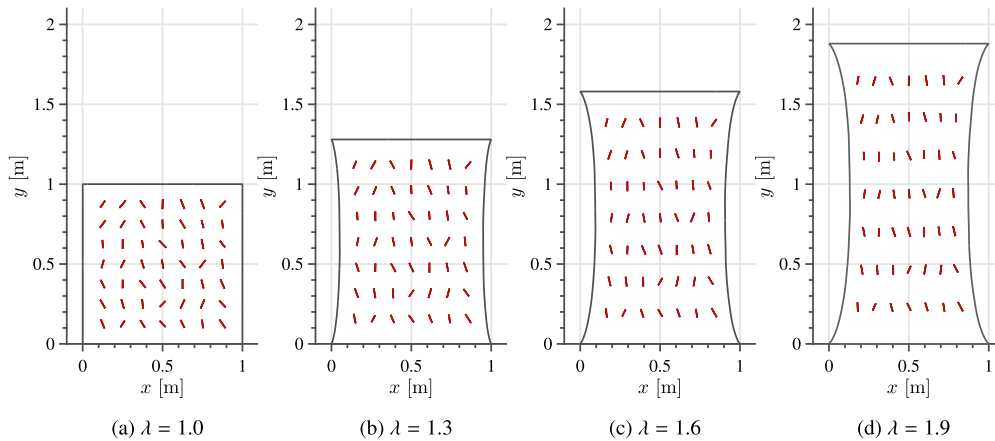


Fig. 17. Deformed configuration of an hyperelastic matrix with 49 randomly oriented anticracks.

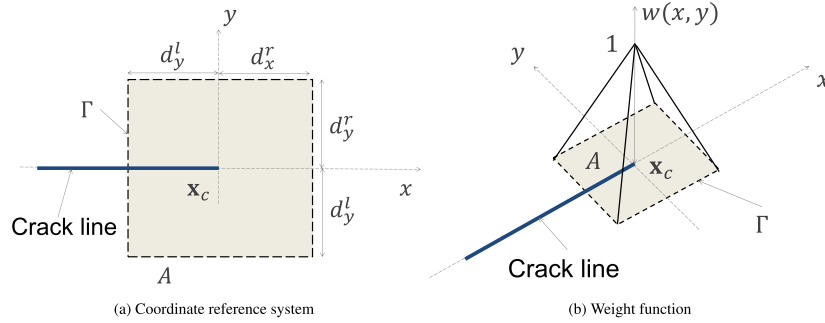


Fig. 18. Domain and weight function w for the J -integral.

inclusions. It is useful to recall that *no remeshing* is necessary, and neither a mesh conforming to the anticrack lines: the distribution of nodes is a regularly arranged grid of 100×100 points. The test in Fig. 16 shows the reorientation of 49 anticracks. Their centers are regularly distributed, but their lengths and initial orientation are randomly chosen. The lengths are drawn from a uniform probability distribution function within an interval of $L/20$ and $L/15$; the initial angles instead from an interval of $\pi/4$ and $3/4\pi$. Once again, Figs. 16 and 17 show the realignment of the anticracks towards the direction of loading ($\pi/2$).

4.8. Remarks on the nonlinear solution

Owing to the strong diagonal dominance of the tangent stiffness matrix in Eq. (55), the method converges rather quickly even for numerous anticracks. For the loads increments of $\Delta\lambda = 0.02$, the solutions converged to a relative residual norm of 0.1% in only a couple of iterations. Even if a sophisticated nonlinear solver, like a *line search*, was available, this option was never activated, and a standard Newton–Rahpson solver sufficed. However, it is advisable to keep the increments short: for *large* increments, the solver might fail due to negative Jacobian.

5. Conclusions

This paper presented a numerical implementation of the *rigid line inclusion* model. A rigid line inclusion, also known as *anticrack*, *line stiffener*, *lamellar inhomogeneity* or *rigid ribbon*, is a kinematic model of a very thin and infinitely stiff reinforcement in an elastic matrix. Using numerical *meshfree* techniques for crack modeling, we presented a straightforward generalization that could prove to be useful for modeling engineered nanocomposites and nano and microstructures occurring in natural systems. Linear elastic matrices were initially considered for ease of validation with existing analytical solutions; results showed rapid convergence to analytical stress intensity factors, for relatively coarse mesh sizes, and very good agreement (apart from the stress singularities) with the theoretical solutions given by Atkinson (1973). Differently from cracks, stress singularities appear also for compressive loadings normal to the inclusion, and for both tensile and compressive loadings parallel to the inclusion: instead, it is *transparent* (no singularity) for a certain combination of biaxial loadings, whose ratio can be predicted from the theory (Atkinson, 1973). The presented method is able to reproduce successfully all these peculiar characteristics of an anticrack. Furthermore, this paper presented extensions to nonlinear elasticity (isotropic Neo-Hookean constitutive law) and multiple anticracks. Firstly, we presented a fiber-reorientation problem under large strains, common in many biological systems, and reported the influence of the orientation of the inclusion with respect to its length, initial inclination and the

stiffness of the matrix. Furthermore, we discussed the influence of two anticracks, and found that there is only slight influence (less than 1°) on the reorientation of very close RLIs. Finally, to prove the power of the approach, we presented an example with 49 anticracks, randomly oriented, and proved that such a complicated system requires no advanced meshing techniques for its treatment.

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Appendix A. Numerical computation of the J -integral

Following Li et al. (1985), the J -integral (Rice, 1968) for single mode loading and a crack that grows straight ahead can be reformulated as in Eq. (15), where A is the domain enclosed by a closed curve Γ oriented anti-clockwise that surrounds the crack tip \mathbf{x}_c , q is a weight function that is zero on Γ and 1 on \mathbf{x}_c (Fig. 18), W is the *strain energy density* that for a linear elastic material can be written as

$$W = \frac{1}{2} \boldsymbol{\sigma} : \boldsymbol{\epsilon} \quad (\text{A.1})$$

The chosen weight function (Fig. 18) is an asymmetric *hat* function given by

$$w(\mathbf{x}, \mathbf{y}) = w_x(x)w_y(y)$$

$$w_x(x) = \left[\mathcal{H}(x + d_x^l) - \mathcal{H}(x) \right] \frac{x + d_x^l}{d_x^l} + \left[\mathcal{H}(x) - \mathcal{H}(x - d_x^r) \right] \frac{-x + d_x^r}{d_x^r}$$

$$w_y(y) = \left[\mathcal{H}(y + d_y^l) - \mathcal{H}(y) \right] \frac{y + d_y^l}{d_y^l} + \left[\mathcal{H}(y) - \mathcal{H}(y - d_y^r) \right] \frac{-y + d_y^r}{d_y^r} \quad (\text{A.2})$$

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